



Considerations for Electronic Equipment Rooms

- Normal Load Density
- Preferable for the room to be in an interior zone.
- Isolation from base building air conditioning.
- Vapour sealing of room



Considerations for Electronic Equipment Rooms

- Normal Load Density
 - Building Fabric Load
 - Equipment Load

- 0.35 ~ 0.85 kW/m²
 - $-0.1 \sim 0.2 \ kW/m^2$
 - 0.15 ~ 0.65 kW/m²

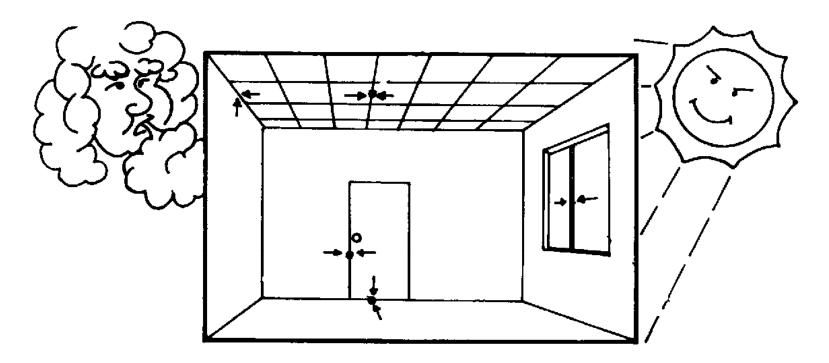
In base building load **sensible 75%** approx latent approx 25% - the sensible load **0.15 kW/m²** equipment load is **all sensible**.

Hence

"SHF" is then 90 ~ 98%



Preferable for the room to be in an internal zone



 Isolation from base building air conditioning. Vapour sealing of room.



Vapour sealing of room

- Walls slab to slab
- Doors well-fitted with seals and open out.
- If on an exterior zone, seal window frames.
- Seal all cable and other penetrations



Vapour sealing of room

A 6m x 10m room with and 2.7m in height $= 162m^3$ which is ~190kg of air.

@24°C 60%RH = 11.3 g/kg or 2.15 kg
@24°C 50%RH = 9.3 g/kg or 1.77 kg
@24°C 40%RH = 7.5 g/kg or 1.42 kg
Differential = **3.8 g/kg** or 722 grams of moisture



Why vapour seal a room

If a 5kW humidifier runs 25% of the time - this is 2,190 hours/annum and at 14c/kWh = \$1,500/annum.

Add additional maintenance to this (replacing consumable items) and it represents over \$2,000 additional costs per annum

Advantage: improved room gradients



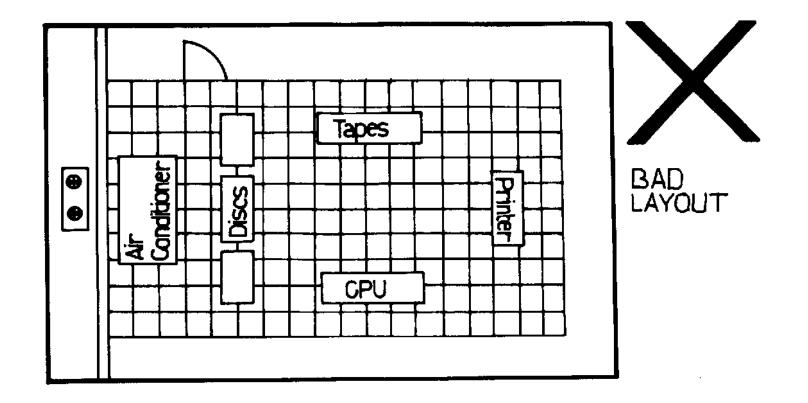
Small Rooms

- Small rooms are more difficult to control than larger, multiple unit, rooms.
 - Because the room has all the constraints (doors, windows, people movement, etc.) of large rooms.
 - Small rooms have less capacity to humidify or dehumidify.

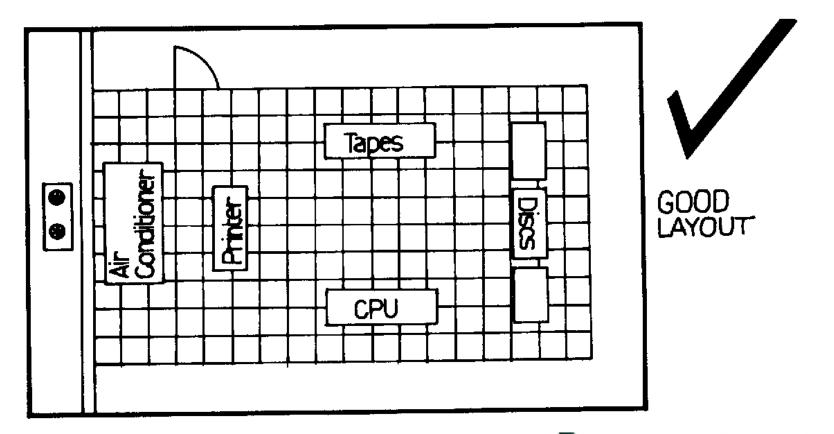
A 5 kW unit with 90% sensible heat factor, has only 0.5 kW of Latent capacity

(can dehumidify/remove only ~0.5 litre of water per hour).

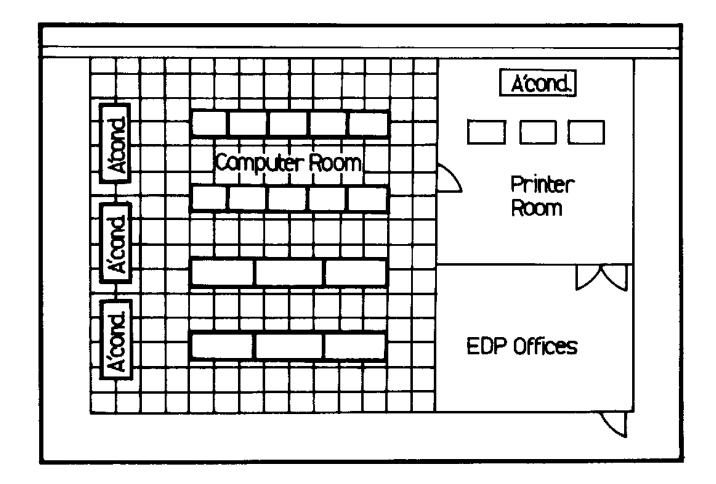








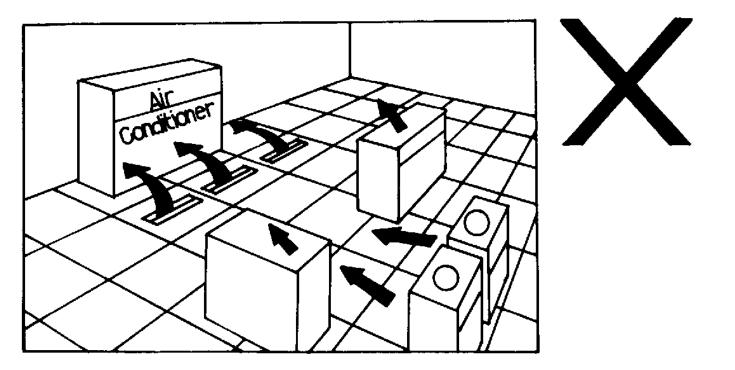






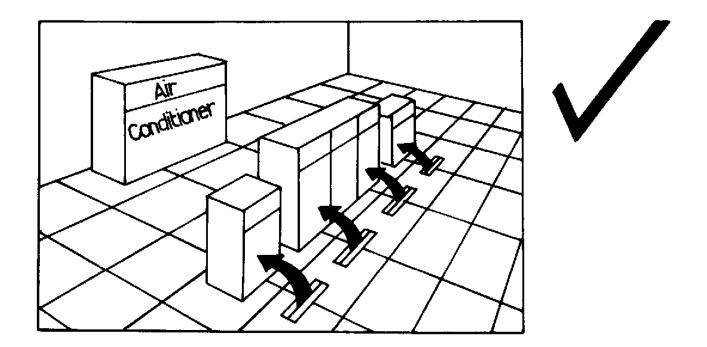
EQUIPMENT_CONFIGURATION

. Avoid floor grilles adjacent to air conditioning equipment otherwise computer heat is not cooled by supply air and bad room conditions result





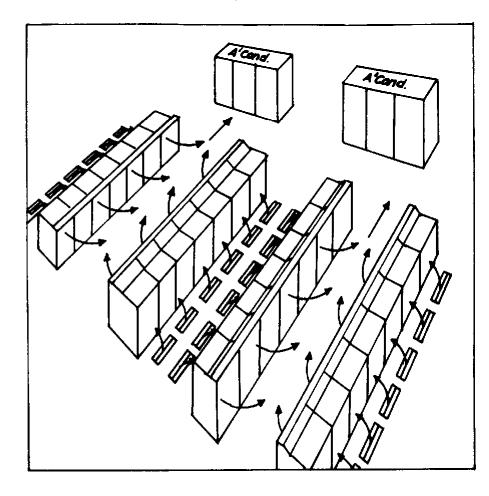
. Locate air conditioning behind heat sources for smaller configurations and position floor grilles to cool individual computer cabinets.





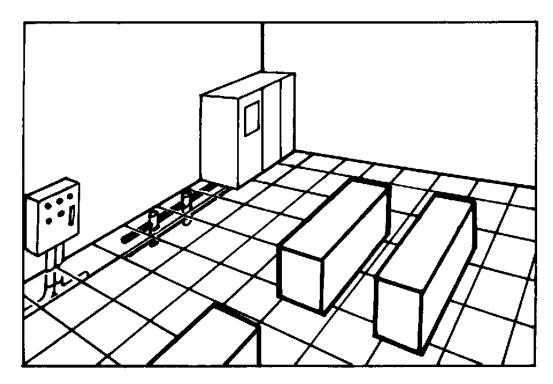
For installations with multiple rows of equipment, locate air conditioning at the ends of the rows to prevent hot air from one row entering adjacent equipment - eg: disc drives

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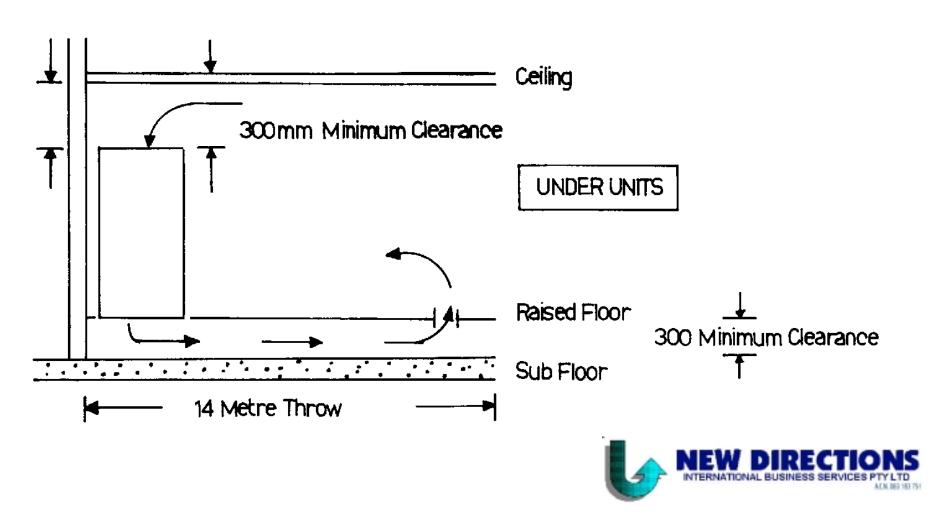


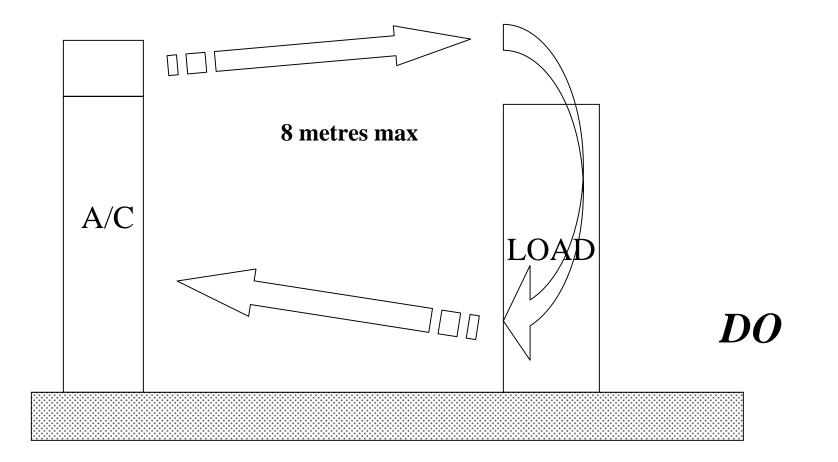
. Install air conditioning power and pipework systems around the perimeter of the room to eliminate interference with the computer operations



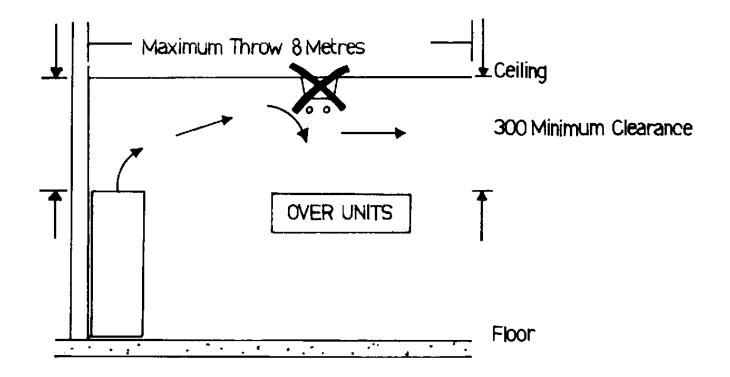


ROOM APPLICATION REQUIREMENTS





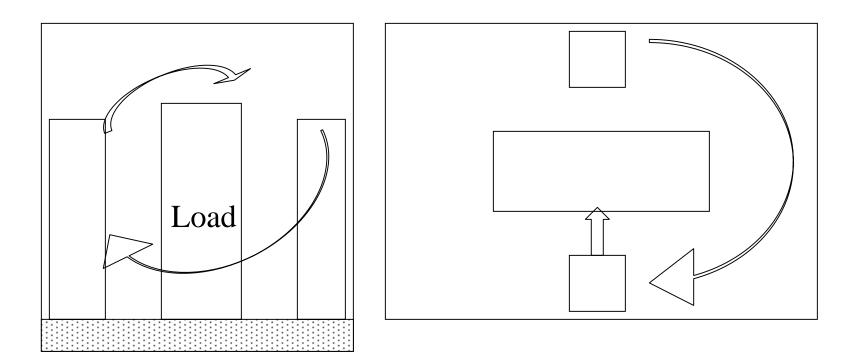




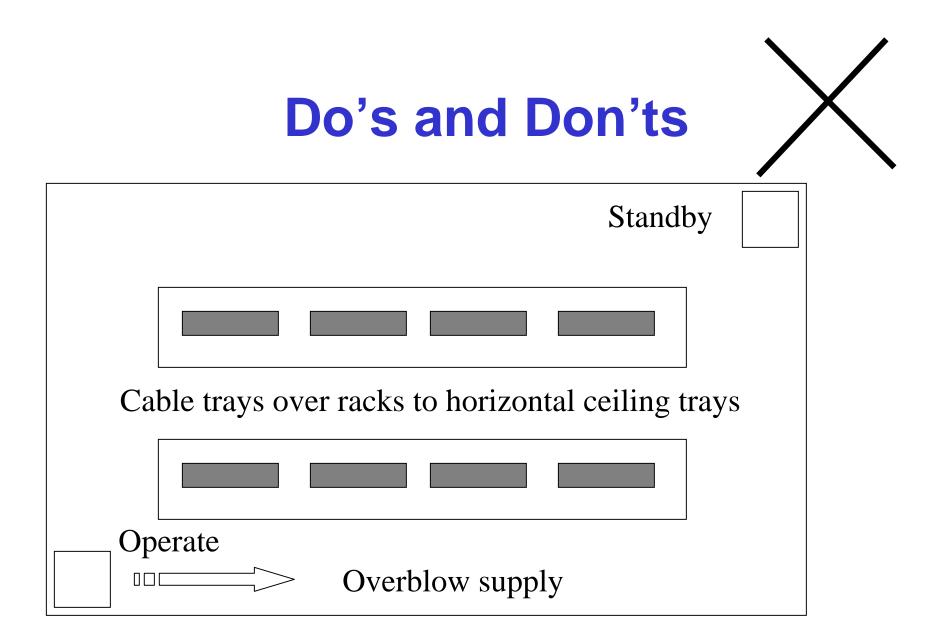
Exposed light fittings in over systems can interrupt airflow and affect room conditions.













Controls response

The air change rate in a room with 0.6 kW/m² with 2.4 m ceiling height and an airflow of 100 l/s per kW of cooling = 90 air changes/hour



Controls response

Changes in Load conditions are reflected at the unit (controls) in **under 1 minute** (delivery)

This requires that the Load calculation plus the equipment are selected carefully so that temperature tolerances can be met.



A room of 60m² with 2.7 m room height (floor to ceiling incl. floor plenum) with a room load of 40kW (0.67 kW/m²) and an airflow of 4000 l/s Selection:

1 x 40 kW 2 circuit unit (operative) plus,

1 x 40 kW 2 circuit unit (standby).

Selected equipment which provides 50 kW Gross total and sensible.



Selected equipment which provides 50 kW Gross total and sensible. Room fabric 0.1 kW/m2 = 6 kW. When fabric load is zero - Load = 34 kW Winter operation Load = 31 kW



With 50 kW 2 step unit

Step 1 = 22 kW (25 kW minus 3 kW fan heat) +Step 2 = 47 kW

The 31 kW cooling requirement is met by the Step 1 continuous cooling and Step 2 cycling 36% on.

When two steps room over-cooled by 16 kW When one step room under-cools by 9 kW

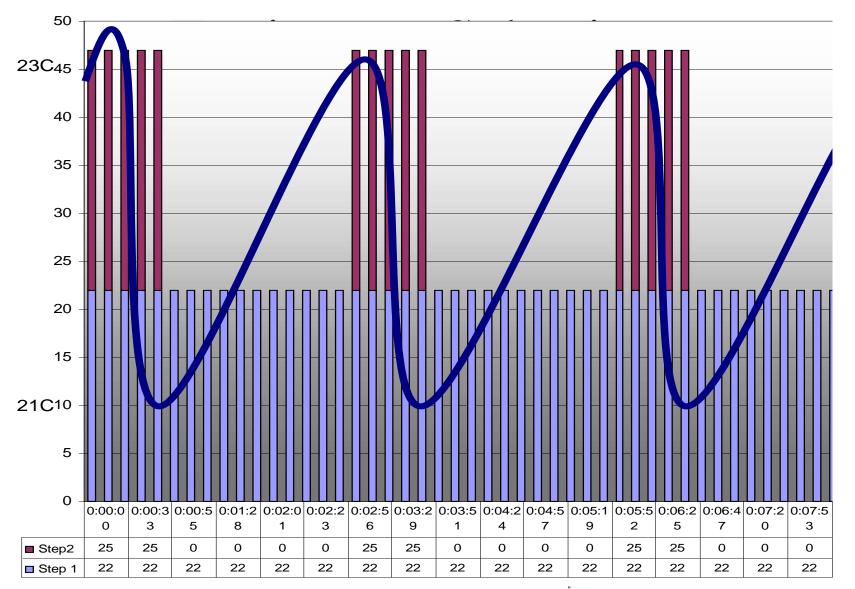


Hence pull-down time on 2 Step cool is $2 \, ^{\circ}C (\pm 2^{\circ}C)$ in 25 seconds + delivery (~20 seconds).

Then time to next cooling call for Step 2 is 44 seconds plus delivery (~ 20 seconds).

Cycles ~ every 2½ minutes or **24 starts per hour.**







Same room / same conditions - With **3 x 26 kW** units selecting **2 operating** and 1 standby.

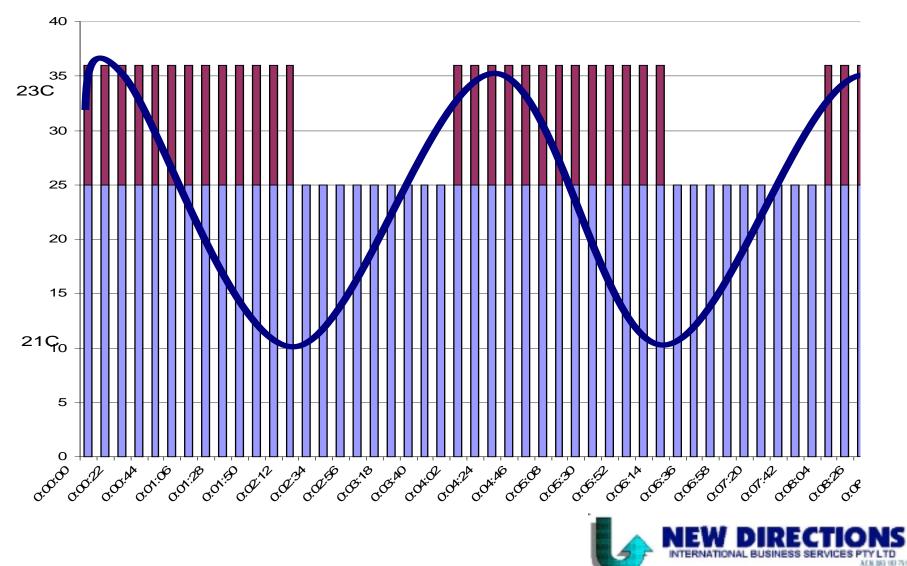
4 available compressors Unit 1 Step 1 & 2 (2 steps) = 24 kW net (26 kW minus 2 kW fan heat)

Unit 2 Step 1 (3rd step) = 11 kW total cooling = 35 kW



Hence compressor Unit 2 Step 1 (3rd step) runs 64% of the time cooling requirement is met by the Step 1 & 2 continuous cooling and 3rd Step cycling 64% on.





Three steps over-cools by 4 kW

Two steps under-cools by 7 kW

Hence pull-down time on 3rd Step is 2°C (±2 °C) in 100 seconds plus delivery (20 seconds).

Then time to next cooling call for 3rd Step is 60 seconds plus delivery (~ 20 seconds).

Cycles ~ every 4 minutes or 15 starts per hour.



Initial Investment Increase:3x 26 kW units = ~\$80,0002x 40 kW units = ~\$70,000Difference (equipment)plus installation-\$ 5,000-\$15,000



Air Cooled Condenser Selection

Select condensers for normal design ambient - not abnormal ambient temperatures:

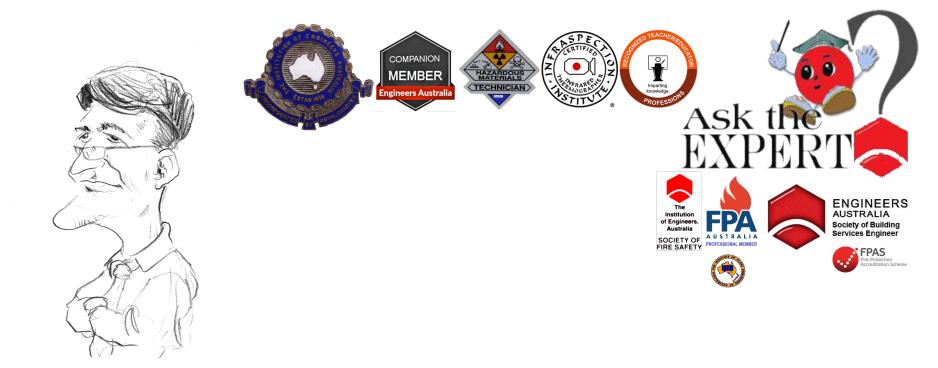
Sydney = **32** ^{**°**}C - **not 40**[°]C

Higher ambient temperatures usually last but a few hours and result in performance reductions of possibly 5% capacity.

These units run 24 hours/day year-through, so oversizing of condensers causes greater problems during low (winter) ambient temperature conditions.

Low condensing pressures force down unit suction temperatures which cause unnecessary dehumidification which then requires the humidifier to operate unnecessarily.





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